INCH-POUND MIL-DTL-868/1A(SH) 14 January 2014 SUPERSEDING MIL-V-868/1(SH) 21 July 1989

DETAIL SPECIFICATION SHEET

VALVES, HYDRAULIC DIRECTIONAL CONTROL, FOUR-WAY, 0-12 GPM

Reinstated after 14 January 2014 and may be used for new and existing designs and acquisitions.

This specification is approved for use by the Naval Sea Systems Command, Department of the Navy, and is available for use by all Departments and Agencies of the Department of Defense.

The requirements for acquiring the product described herein shall consist of this specification sheet and MIL-DTL-868.

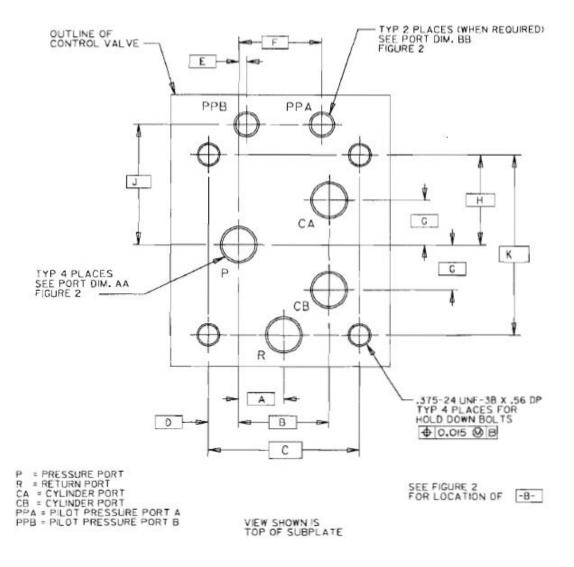
1. GENERAL

1.1 <u>Classification</u>. Directional control valves covered by this specification sheet shall be designated using Part or Identifying Numbers (PIN) as follows (see <u>table V</u> for examples of PINs):

М	868	/1	-	WW	Т	YYY	ZZ	Х
Prefix to indicate a Military specification	Specific ation number	Applicable specification sheet number (see 1.1.1)		Left side operator (see 1.1.2)	Spool type (see 1.1.3)	Spool flow paths (see 1.1.4)	Right side operator (see 1.1.5)	Fatigue (see 1.1.6)

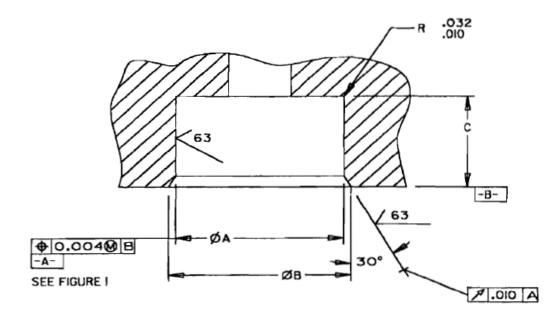
1.1.1 <u>Applicable specification sheet number</u>. The applicable specification sheet number designates the valve's performance characteristics (flow range, pressure losses, leakage, etc.) and mounting interface configuration. <u>Figures 1, 2</u>, and <u>3</u> define the mounting interface configuration and maximum dimensions for valves with various types of operators. Control valves covered by this specification sheet shall be of the integral slide and sleeve type.

1.1.2 Left side valve operator. The left side valve operator shall be designated by a two-character code. The left hand operator shall be designated as the actuator shown to the left of the spool symbol. On the actual valve, this corresponds to the actuator closest to the return port. For valves operated by a handle or lever, the two-character code shall be in accordance with table I. The first character designates the type of operator in addition to the handle. The second character is a number (0 to 7) and indicates the positions, if any, in which the valve spool may be latched or detented. For valves without an operating handle, the left side operator shall be designated by a two-character code in accordance with table II. The first letter designates the primary operator and the second letter identifies secondary operator(s), if any. See 1.1.7 for special operator configurations not covered by tables I and II.



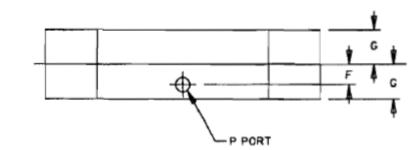
Dimensions in inches				
А	В	С	D	Е
0.750	1.500	2.500	0.500	0.125
F	G	Н	J	K
1.375	0.750	1.500	2.000	3.000

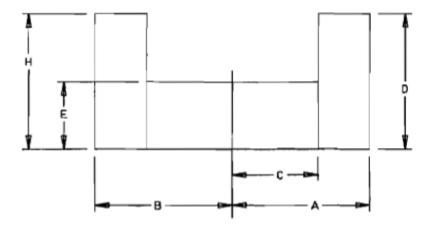
FIGURE 1. Valve mounting configuration.



Counterbore dimensions in inches					
Port	ØA	ØB	С		
AA	0.550/0.552	0.580/0.610	0.440/0.446		
BB	0.360/0.361	0.390/0.420	0.338/0.344		

FIGURE 2. Counterbore dimensions.



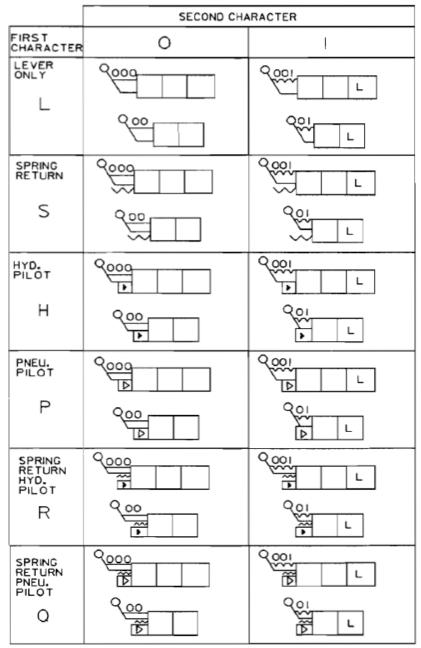


DIMENSIONS IN INCHES

	А	В	С	D	Е	F	G	Н
	max	max	min	max	max	max	max	max
Double solenoid	7.50	7.50	2.70	6.50	4.50	0.77	1.63	6.50
Single solenoid	7.50	9.00	2.70	6.50	4.50	0.77	1.63	
Solenoid or hyd. pilot with lever override	10.12	8.38	2.70	12.00	4.50	0.77	1.63	6.50
Lever operation	10.12	8.50	2.70	8.62	4.50	0.77	1.63	
Mechanical	7.50	7.50	2.70	4.50	4.50	0.77	1.63	

FIGURE 3. Valve envelope.

TABLE I. <u>Valve operators-lever</u>.



SYMBOLS 'I' AND 'L' INDICATES THE POSITION THAT THE VALVE SPOOL IS LATCHED OR DETENTED.

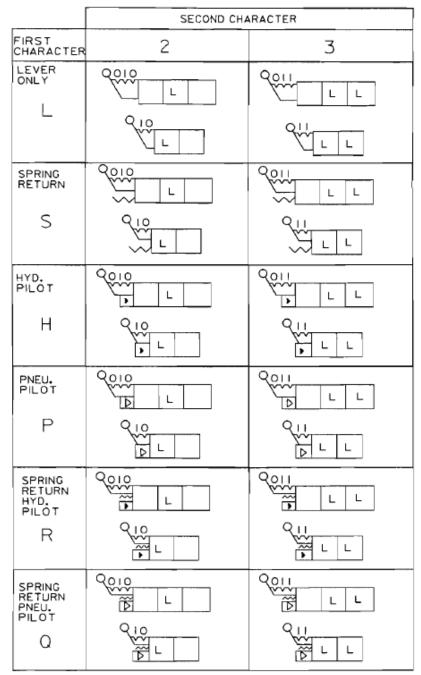


TABLE I. Valve operators-lever Continued.

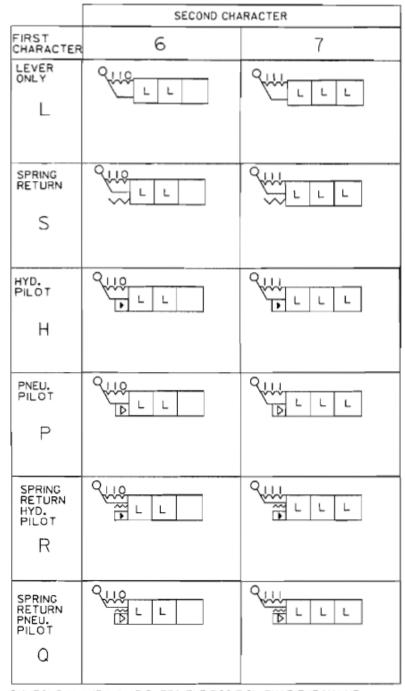
SYMBOLS "I' AND "L' INDICATES THE POSITION THAT THE VALVE SPOOL IS LATCHED OR DETENTED.

SECOND CHARACTER FIRST CHARACTER 5 4 LEVÉR ONLY Q100 Q 101 L L L L 9,100 QUOI SPRING RETURN L L L S 9,01 9,100 HYD. PILOT L L L . Н 9,100 9,101 PNEU. PILOT L L L Þ Þ Ρ 9,00 9,101 SPRING RETURN HYD. PILOT õ ñ L L L R 9,100 9,101 SPRING RETURN PNEU. PILOT ñ 而上 L L Q

TABLE I. Valve operators-lever Continued.

SYMBOLS 'I' AND 'L' INDICATES THE POSITION THAT THE VALVE IS LATCHED OR DETENTED.

TABLE I. Valve operators-lever – Continued.



SYMBOLS "I' AND "L' INDICATES THE POSITION THAT THE VALVE SPOOL IS LATCHED OR DETENTED.

FIRST	SECO	ND LETTER (ADDI	TIONAL OPERATO	
FIRST	X (NONE)	M (MANUAL)	S (SPRING)	T (SPRG RET MAN OVRRID)
CAM	Ъ	NA	۶P 	NA
ELECTRICAL				
HYDRAULIC				
MANUAL		NA		NA
PNEUMATIC			~	
SPRING S	~	NA	NA	NA
TWO-STAGE		Ľ ∠ ,		
WITHOUT OPERATOR X		NA	NA	NA

TABLE II. Valve operators.

NOTE:

1. Only listed combinations may be used. NA = Not available.

Example: ES designates a solenoid operated valve with return spring.

H2 designates a manually operated valve with a hydraulic pilot override which is latched or detented in the center spool position.

1.1.3 <u>Spool metering or adjustment features</u>. The spool metering or adjustment features are designated by a single character as follows:

- Standard valve with no metering or adjustable flow limiters.
- M or N Metering valve where flow is proportional to spool displacement. The maximum fluid flow can be limited by adjustable spool stroke limiters (see 2.1, 2.1.1, and 2.1.2).
- V or A Adjustable valve where fluid flow can be limited by spool stroke limiters (see 2.1 and 2.1.1).

1.1.4 <u>Spool flow paths</u>. Each valve position flow path shall be designated by a number (or character) in accordance with <u>table IV</u>. A two-position valve is designated by two numbers (or characters) and a three-position valve is designated by three numbers (or characters). Use of more than three characters to designate spool flow paths shall require approval from the Naval Sea Systems Command. See <u>table V</u> for examples of part numbers.

1.1.5 <u>Right-side valve operator</u>. The right-side valve operator shall be designated by a two-character code in accordance with <u>table II</u>. The first letter designates the primary operator and the second letter identifies secondary operator(s), if any. See 1.1.7 for special operator configurations not covered by <u>tables I</u> and <u>II</u>.

Example: EM designates a solenoid operated valve with manual override.

SX designates a return spring only.

1.1.6 <u>Fatigue grade</u>. Fatigue rated valves shall conform to the fatigue rated test requirements of MIL-DTL-868. A fatigue rated valve is suitable for use in a non-fatigue rated application. The single letter designates the fatigue requirements of the valve as indicated in <u>table III</u> below:

Non-fatigue rated designation	Fatigue rated designation
В	G

TABLE III. Fatigue grade.

CODE	SYMBOL	CODE	SYMBOL	CODE	SYMBOL
0	CA CB	I	CA CB	2	CA CB
3	CA CB	4	CA CB	5	
6	CA CB	7		8	CA CB
9	CA CB	Z	CA CB	к	
L	CA CB				

TABLE IV. Spool flow path.

NOTE:

1. Port orientation shall be as defined herein unless otherwise approved by the Naval Sea Systems Command.

1.1.7 <u>Special configuration</u>. Where the required valve configuration cannot be described by the standard designators in <u>tables I</u>, <u>II</u>, and <u>IV</u>, the part number designators will be assigned by the Naval Sea System Command following approval of the desired configuration.

1.2 <u>Mounting configuration</u>. The mounting configuration of valves shall be as shown on <u>figure 1</u>.

1.2.1 Fluid connection. The quill or sealing sleeve configuration shown on figure 2 shall be used.

1.2.2 <u>Pilot ports</u>. Ports PPA and PPB, shown on <u>figure 1</u>, are used with valves that require external hydraulic pilot supply to shift the spool. These are valves that have operators which are identified by the letter or character codes H, P, R, and Q from <u>tables I</u> and <u>II</u>.

1.3 <u>Envelope dimensions</u>. Unless otherwise specified (see 6.2 of MIL-DTL-868), the valve assembly shall not exceed the limitations of the envelope dimensions listed on <u>figure 3</u>.

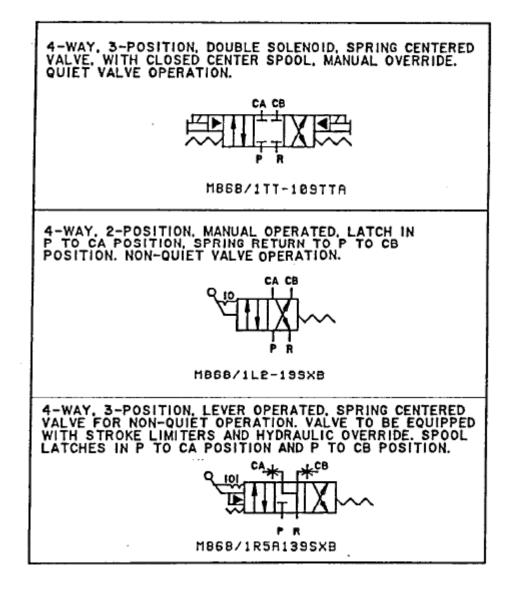
1.4 <u>Solenoids</u>. The solenoids shall be of the direct current type with a rectifier to convert the incoming alternating current to direct current.

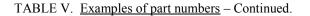
1.5 <u>Hydraulic fluid</u>. The valve shall meet the fluid compatibility requirements specified in MIL-STD-2193, including the immersion test for nonmetallic materials. Fluid in accordance with MIL-PRF-17331 (2190 TEP) or MIL-PRF-17672 (2075-T-H) may be used for the immersion test.

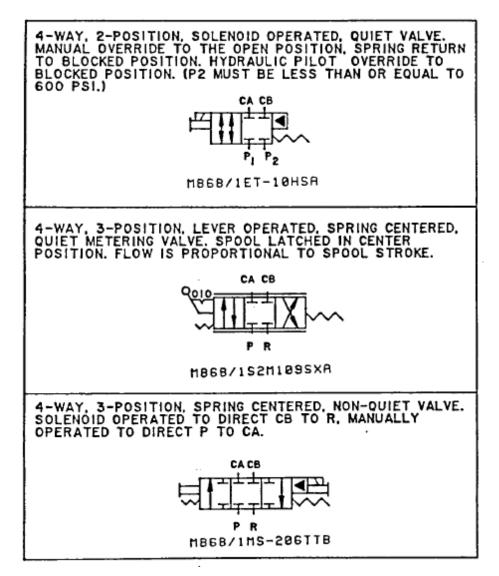
1.6 <u>Test fluid and viscosity</u>. The test fluid shall be either of the two fluids specified in 1.5. Unless otherwise specified for the specific test, the fluid temperature shall be controlled to meet the viscosity requirements identified below. See MIL-DTL-868 for guidance regarding fluid viscosity at various test temperatures.

- a. For leakage test, the maximum fluid viscosity at atmospheric pressure shall be 75 centistokes.
- b. For pressure drop tests, hydraulic actuation tests, and solenoid actuation tests, the fluid viscosity at atmospheric pressure shall be between 75 and 97 centistokes.
- c. For noise testing, the fluid viscosity at atmospheric pressure shall not exceed 50 centistokes.









2. PERFORMANCE

2.1 Flow rate. Valves covered by this specification sheet shall have the following ranges found in table VI:

Spool type (1.1.3)	Flow rate (gallons per minute (GPM))			
-, M and A	0.5 - 12.0			
-, M and A	0.5 - 6.0 1/			
N and V	0.03 - 1.5			
NOTE:				
^{1/2} Flow rate range only applicable when noise requirements are specified in the procurement documentation (see 6.2 of MIL-DTL-868).				

ΓABLE VI.	Flow rate ranges.

2.1.1 <u>Flow control adjustment</u>. Valves with flow control adjustments shall be capable of restricting flow to the minimum values listed in 2.1 at a minimum differential pressure of 2,200 pounds per square inch gauge (psig) and a viscosity of 50 centistokes maximum at atmospheric pressure.

2.1.2 <u>Differential pressure at rated flow</u>. The differential pressure drop across a valve with the designated spool style at the maximum rated flow listed in 2.1 shall be as follows:

- a. For valves that utilize flow paths 1 and 9 from <u>table IV</u>, the differential pressure across the valve in that position shall not exceed 500 pounds per square inch differential (psid). The pressure differential shall be measured from the pressure port (P) to the return port (R) with the cylinder ports (CA and CB) connected by a piping loop.
- b. For valves that utilize flow paths 2, 3, 5, 6, and Z from <u>table IV</u>, the differential pressure across the valve in that position shall not exceed 250 psid. The pressure differential shall be measured from either the P or R port to either the CA or CB port. The remaining port is blocked.
- c. For valves that utilize flow paths 4, 7, and 8 from <u>table IV</u>, the differential pressure across the valve in that position shall not exceed 250 psid. The pressure differential shall be measured from one of the ports to the other.
- d. For metering valves (spool type M and N), the valve flow characteristics versus spool displacement shall be within the requirements as specified in <u>table VII</u>.

Percent	Percent rated flow at 500 psid				
spool stroke ^{1/}	Maximum	Minimum			
5	5	0			
10	10	2			
20	20	5			
40	40	16			
60	60	32			
80		50			
100		100			
NOTE: ^{1/} Percent spool stroke can be considered the range in which flow occurs, not the total spool stroke which includes spool/sleeve overlap.					

TABLE VII. Metering valve flow.

2.2 <u>Solenoid actuation</u>. The solenoid actuation rate specified in MIL-DTL-868 and <u>table VIII</u> shall be met with the valve at ambient room temperature conditions:

Condition	Supply pressure (psig)	Return pressure (psig)	Shift time (milliseconds)
1	2,500±100	100±40	500 max
2	1,200 max	100 min	1,500 max

TABLE VIII. Solenoid actuation rate.

2.3 <u>Leakage</u>. For all <u>table IV</u> spool flow paths except paths 4 and 8, the maximum allowable leakage from the P port to the R port with the CA and CB ports blocked shall be 0.10 ounces/minute (3 mL/minute). For path 8, the maximum allowable leakage from the CA and CB ports to the R port with the P port blocked shall be 0.10 ounces/minute (3 mL/minute). Leakage across the restricted flow path (orifice) for spool flow paths J and K shall not be less than 0.14 ounces/minute (4 mL/minute) nor more than 0.51 ounces/minute (15 mL/minute).

2.4 <u>Noise</u>. The structureborne noise tests for all applicable valves (see 3.5.5 of MIL-DTL-868) shall be performed in accordance with the requirements specified in MIL-DTL-868 when tested under the following conditions.

2.4.1 Steady state test conditions.

a. Non-metering valves and valves without stroke limiting adjustment.

Flow - Maximum flow (see 2.1).

Supply pressure - As required to obtain maximum flow.

Return pressure - 60 psig maximum.

b. Metering valves and valves with stroke limiting adjustments shall be tested for Runs 1, 2, 3, and 5 in <u>table IX</u>. Return pressure shall be 60 psig maximum for all tests. Record values as identified in <u>table IX</u>.

Run	Spool position % of full stroke	Flow rate	Differential pressure (psid)
1	As required (record)	Minimum rated	2,200+100-0
2	As required (record)	50%	2,200+100-0
3	As required (record)	Maximum rated	2,200+100-0
4	As required (record)	20%	2,200+100-0
5	100%	Maximum rated	\leq 500 (record)
6	As required (record)	Highest Actual	2,200+100-0

TABLE IX. Steady state test conditions for metering valves and valves with stroke limiting adjustment.

2.4.2 Transient test conditions.

2.4.2.1 <u>Blocked line test</u>. This test shall be conducted in accordance with MIL-DTL-868.

2.4.2.2. <u>Shift at rated flow</u>. For non-metering valves without stroke limiting adjustments, steady state maximum rated flow as required in 2.4.1.a shall be achieved. All valves shall meet the shift time and test positions of MIL-DTL-868. For metering valves and valves with stroke limiting adjustments:

- a. First article testing shall be conducted for the conditions identified in Runs 2, 3, 4, and 6 of table IX.
- b. Conformance testing shall be conducted for the conditions identified in Run 6 of table IX.

First article testing at flow rates above the highest actual flow shall be for information only.

2.5 <u>Maintainability</u>. The requirements specified in MIL-DTL-868 shall apply.

2.6 <u>Flow sensitivity to viscosity</u>. The viscosity sensitivity of the valve is limited to the parameters tested below. Their requirements must be met for the valve's entire adjustment range at a constant differential pressure. A valve initially set up in accordance with condition (a) below shall meet conditions (b) and (c).

- a. The flow rate passed through the valve with the supply fluid of a viscosity of 75±5 centistokes (MIL-PRF-17331 [2190 TEP] at 110 °F [43 °C]) shall be considered 100 percent of set flow.
- b. With a supply fluid of a viscosity of 370 centistokes minimum (MIL-PRF-17331 [2190 TEP] at 60 °F [16 °C]), the valve with a pressure drop, as specified in <u>table X</u>, shall pass no less than the percentage, as specified in <u>table X</u>, of set flow as defined under condition (a). Non-metering valves without stroke limiting adjustment are exempt from the 1,500 and 2,200 psid tests specified below in <u>table X</u>.

Pressure drop for valves with flow paths 1 and 9 per <u>table IV</u> (psid)	Pressure drop for valves with flow paths 2 through 8 and Z per <u>table IV</u> (psid)	Percentage of set flow
500	250	30%
1,500	1,500	50%
2,200	2,200	60%

TABLE X.	High viscosity test conditions.

c. With a supply fluid of a viscosity of 38 centistokes maximum (MIL-PRF-17331 [2190 TEP] at 140 °F [60 °C]), the valve with a pressure drop of 500 psid, for valves of flow paths 1 and 9 per <u>table IV</u>, and 250 psid, for valves of flow paths 2 through 8 and Z per <u>table IV</u>, shall pass no greater than 130 percent of set flow as defined by condition (a).

3. CHANGES FROM PREVIOUS ISSUE. Marginal notations are not used in this revision to identify changes with respect to the previous issue due to the extent of the changes.

Preparing Activity: Navy – SH (Project 4810-2013-002)

NOTE: The activities listed above were interested in this document as of the date of this document. Since organizations and responsibilities can change, you should verify the currency of the information above using the ASSIST Online database at <u>https://assist.dla.mil</u>.